

Feature Article

Application

Real-time Fuel Consumption Measurement Using Raw Exhaust Flow Meter and Zirconia AFR Sensor

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The improvement of fuel efficiency is one of the most important issues in the R&D of powertrain system. The real-time fuel consumption can be determined by AFR (air-to-fuel ratio) and raw exhaust gas flow rate and can be easily obtained without delay time by the in-situ measuring devices which can be installed at the same location. Integrated fuel consumption by this method showed a good correlation with that by the carbon balance method. On the other hand, when a fuel-cut is operated, the difference in transient behavior of the fuel consumption has been also observed due to the response time difference between these two methods. The result suggests that this method has a large potential for measuring the real-time fuel consumption.

Introduction

Direct measurement of exhaust flow rate had been one of the difficult challenges in exhaust gas analysis and evaluation. Direct measurement of exhaust gas had not been so urgently needed in the past, since the regulating methods, which are used in evaluation of light/medium duty vehicles for conformance to the exhaust gas regulation, adopt the constant volume dilution sampling (CVS) method which requires no exhaust gas flow rate. However, the concern that the intermittent operation of engine while driving would cause errors in measurements by CVS method was raised as hybrid vehicles and idle stop vehicles have become more popular. While improvement in CVS method itself was proposed by intermittently operating the sampling itself in synchronicity with this intermittent operation^[1], direct mass measurement method which is calculated by direct measuring of the exhaust gas flow rate and concentration has been gaining attention especially for research and development purposes. With such circumstances in the background, the authors developed an ultrasonic exhaust flowmeter and proposes to the customers the direct mass measurement in combination with direct exhaust gas analyzer as a measure to improve the engine/vehicle development efficiency.

In this study, we propose a real-time fuel consumption measurement using the exhaust gas flow rate and air-to-

fuel ratio (AFR) as a measurement application of this exhaust gas flowmeter. This is a new measurement proposed by HORIBA. An advantage of this measurement method is that the fuel consumption can be calculated by only measuring the exhaust gas and AFR the direct mass measurement in combination with direct exhaust gas measurement to improve the engine/vehicle development efficiency, including completed vehicles. In general, fuel consumption is measured by connecting a fuel flowmeter to the fuel piping from the fuel tank to the engine when evaluation is conducted on engine bench^[2]. On the other hand, it is difficult to add the fuel flowmeter and a special piping for it to the fuel piping system in tests on completed vehicles. Although it is possible to measure the fuel consumption without any processing of the vehicle when carbon balance method adopting the CVS method is used, it does not deliver sufficient response time for rapid changes in fuel consumption rate under transient states due to its principle. The authors therefore examined “exhaust gas flow rate/AFR method” as a method capable of measuring fuel consumption with response time sufficient for checking the transient behaviors in a simpler method than these conventional methods during automobile research and development. This article reports on its concept, configuration of devices and the results of its evaluation in comparison with the conventional method.

Indirect Methods to Calculate the Fuel Consumption

As methods to indirectly calculate the real-time fuel consumption based on measurement values other than fuel flow rate, the conventional carbon balance method and the method the authors examined in this study (exhaust gas flow rate/AFR method) are described.

Fuel consumption by carbon balance method

The carbon balance method calculates the total carbon content based on the theory that the total carbon mass in the fuel consumed by the engine and the total carbon mass in exhaust gas are equal. That is, the concentrations of exhaust gas components containing carbon (CO₂, CO, and HC) are measured by the CVS method to be converted into emitted mass and calculate the total carbon mass. Then the mass of fuel eventually consumed is calculated. While the bag method to measure the concentration after accumulating a certain volume of diluted exhaust gas in the sampling bag is generally used in CVS method, it is possible to learn the real-time emission mass of each component and calculate the fuel consumption continuously by using a dilute continuous measurement method (dilute stream method) instead of the bag method.^[3] Equation 1 shows the method to calculate the fuel consumption in real time by carbon balance method.

$$F_{CB}(t) = \frac{1}{R_{CWF}} \times \left(\frac{M_C}{\alpha_{exh} \times M_H + M_C} \times HC_{MASS}(t) \right) + \frac{M_C}{M_{CO}} \times CO_{MASS}(t) + \frac{M_C}{M_{CO_2}} \times CO_{2MASS}(t) \dots \dots \dots (1)$$

Here, $F_{CB}(t)$ indicates the real-time fuel consumption at time t , R_{CWF} the rate of carbon mass in fuel, $HC_{MASS}(t)$, $CO_{MASS}(t)$ and $CO_{2MASS}(t)$ the momentary emission of CO, CO₂, and HC at time t , respectively, M the molar mass for each component, and α_{exh} the average hydrogen-carbon atomic ratio for HC in exhaust gas.

Fuel consumption based on exhaust gas flow rate and AFR

Next, the method to calculate the fuel consumption based on exhaust gas flow rate and AFR is described.

The air-to-fuel ratio (AFR) is the mass ratio between air and fuel supplied to the engine, and is expressed as shown in the following equation^[4]:

$$AFR(t) = \frac{q_{maw}(t)}{q_{mf}(t)} = \frac{q_{maw}(t) - q_{mf}(t)}{q_{mf}(t)} \dots \dots \dots (2) = \frac{q_{vev}(t) \times \rho_{ew} - q_{mf}(t)}{q_{mf}(t)}$$

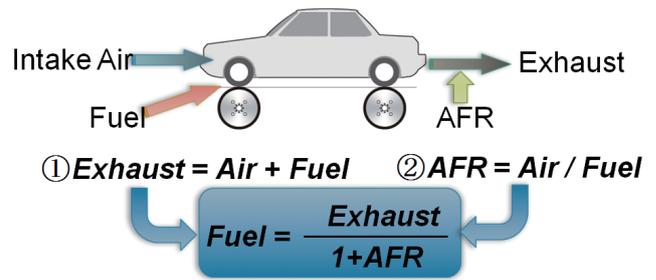


Figure 1 Principle of fuel flow measurement by exhaust gas flow and air-to-fuel ratio.

Here, $AFR(t)$ indicates the air-to-fuel ratio at time t , $q_{maw}(t)$ the mass flow rate for supplied air, $q_{mf}(t)$ the fuel consumption, $q_{mev}(t)$ the mass flow rate for exhaust gas, $q_{vev}(t)$ the volume flow rate for exhaust gas, and ρ_{ew} the exhaust gas density.

By modifying Equation 2, the equation to calculate the fuel consumption using the exhaust gas volume flow rate, exhaust gas density and AFR can be obtained.

$$q_{mf}(t) = \frac{q_{vev}(t) \times \rho_{ew}}{AFR(t) + 1} \dots \dots \dots (3)$$

Although the exhaust gas density changes by several %, it changes little in the lean range and thus the effect of using a constant value can be considered practically negligible. In addition, it is known that the concentration of each component can be estimated by supposing the exhaust gas combustion reaction formula and properly and using the AFR value, and thus measurement accuracy can be improved by calculating the exhaust gas density based on component concentrations.

It may be possible to measure the exhaust gas flow rate and AFR which are the measured parameters of this “exhaust gas flow rate/AFR method” with high response time at engine exhaust pipe as described later^[5]. Therefore, it is expected that it will have the advantage that the results are resistant to the response delay caused by sampling or measuring instruments. However, caution is necessary under conditions where changes in volume tend to occur due to condensation in the pipe up to the exhaust gas flowmeter, including immediately after engine startup (cold start), as the intake air flow rate is calculated using the exhaust gas flow rate.

Configuration of the Measuring Instrument

A newly developed ultrasonic flowmeter was used as the exhaust gas flowmeter. Since this method allows measurements in exhaust pipe with high response time, it

is optimal for this measurement. In addition, a directly inserted zirconia (ZrO₂) sensor which can also be installed directly in the exhaust pipe was used for AFR measurement. These measuring instruments cause extremely small pressure loss through installation and thus the load on the engine can be neglected. The details are described as follows:

Ultrasonic exhaust gas flowmeter

Figure 2 shows the structure of the ultrasonic flowmeter. The ultrasonic transducers are installed with an angle on opposite sides of the piping through which the measurement subject gas flows. These transducers comprise mainly of piezoelectric elements and are capable of converting the electric signal into mechanical vibration. The resonance frequency for these elements is designed in an appropriate ultrasonic frequency band. When voltage pulse for resonance frequency is applied on these transducers, the ultrasonic pulse is oscillated by piezoelectric effect. The ultrasonic pulse propagates through the gas in pipe, reaches the transducer on the other side and become converted into electric signal again to be stored in CPU board as a waveform shown in Figure 3.

If there is gas flow inside the piping, the time for the ultrasonic pulse to propagate is affected by the flow. The propagation time for flows in upstream and downstream directions are expressed as follows, respectively:

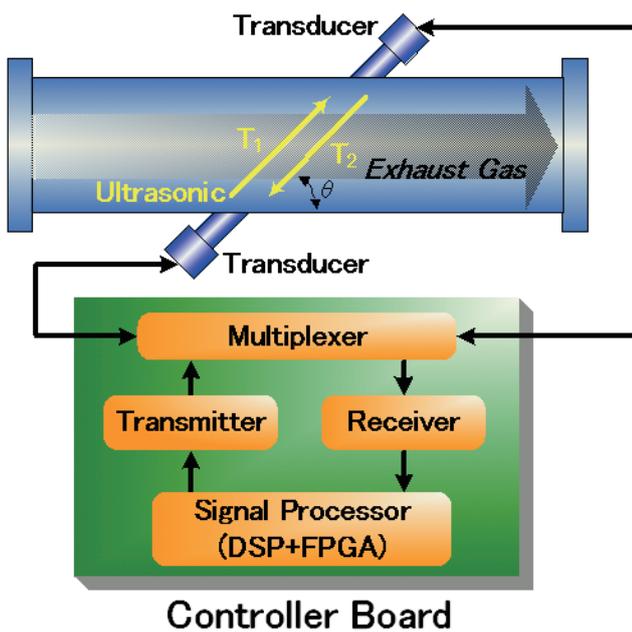


Figure 2 Principle of ultrasonic exhaust gas flow meter

$$T_1 = \frac{L}{c(t) + v(t) \cos\theta} \dots\dots\dots (4)$$

$$T_2 = \frac{L}{c(t) - v(t) \cos\theta} \dots\dots\dots (5)$$

Here, T_1 indicates the propagation time in downstream direction [s], T_2 the propagation time in upstream direction [s], L the distance between ultrasonic transducers [m], c the velocity of sound, v the velocity of gas, and θ the angle of ultrasound propagation.

Equation 4 and 5 can be modified as Equation 6 and 7, respectively.

$$c(t) = \frac{L}{T_1} - v(t) \cos\theta \dots\dots\dots (6)$$

$$c(t) = \frac{L}{T_2} + v(t) \cos\theta \dots\dots\dots (7)$$

When the term for velocity of sound $c(t)$ is deleted from Equation 6 and 7, Equation 8 that indicates the gas flow velocity is obtained.

$$v(t) = \frac{L}{2 \cos\theta} \times \left(\frac{1}{T_1} - \frac{1}{T_2} \right) \dots\dots\dots (8)$$

As shown in Equation 8, the gas flow velocity calculated by this method does not depend on the sound of velocity. That is, the gas flow velocity calculated is not influenced even when gas density changes due to composition change. The exhaust gas volume flow rate (standard condition) can be calculated based on this exhaust gas flow rate and pipe diameter.

$$q_{vev}(t) = k_{profile} \times A \times v(t) \times \frac{T_0}{T(t)} \times \frac{P(t)}{P_0} \dots\dots (9)$$

Here, q_{vev} indicates the exhaust gas volume flow rate (converted for standard condition), $k_{profile}$ the correction coefficient, A the pipe surface area, T_0 the standard temperature, T the exhaust gas temperature, P the absolute pressure for exhaust gas, and P_0 the standard absolute pressure.

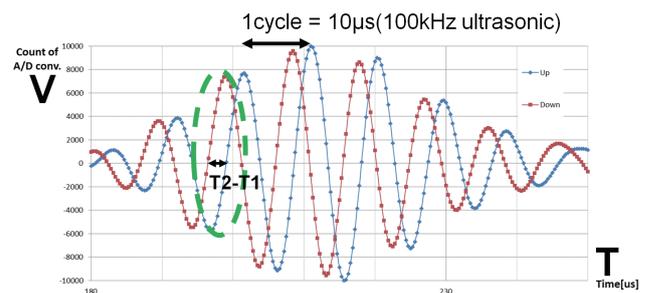


Figure 3 Ultrasonic signal wave form

Furthermore, coefficient $k_{profile}$ is used to correct the effects of gas flow velocity and temperature distribution within the pipe. It is determined by comparison with reference flowmeters such as smooth approach orifice (SAO) flowmeter.

In this study, we used a special ultrasonic flowmeter for engine exhaust gas with the presumption to measure the exhaust gas flow rate in exhaust pipe. Since this device uses a unique ultrasonic transducer, measurements can be taken even when the gas temperature is high. It also has the advantage to reduce the effect of the ultrasonic propagation time caused by pipe stain as the propagation time difference is measured. As the flow rate response converted for standard condition also depends on exhaust gas temperature response, the temperature sensor with sharp response to gas temperature was adopted.

Zirconia AFR sensor

Figure 4 shows the structure of the zirconia AFR sensor, and Figure 5 a photograph of an example of this sensor. The measurement subject gas diffuses through the space inside (diffusion chamber) from diffusion porosity from the sensor surface. The sensing cell block has electrodes formed on both sides of the zirconia solid electrolyte, and monitors the potential difference generated due to O_2 concentration difference between the reference air side and diffusion chamber side. The pump cell block has the function to transfer O_2 via the zirconia solid electrolyte as voltage is applied from an external source. The voltage of the pump cell block is controlled so that the potential difference monitored by the sensing cell block is stable at an equivalent level to the theoretical AFR. That is, it draws out the excessive O_2 entering the diffusion chamber under lean condition and draws in the O_2 to burn the CO, H_2 , and HC inside the diffusion chamber under rich condition. The current proportional to the amount of O_2 transferred (pumping current) flows through the pump cell. The pumping current is expressed as shown in Equation 10:

$$I_p = \frac{n \times F \times S \times P}{R \times T \times L} \times D_{O_2} \times C_{O_2} \quad \dots \dots \dots (10)$$

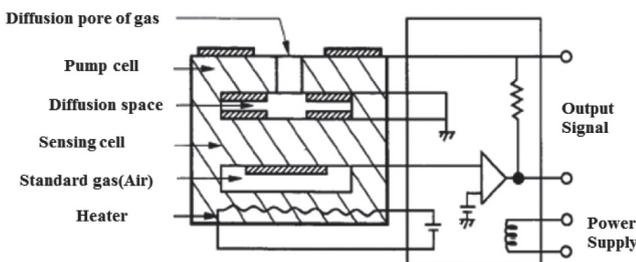


Figure 4 Configuration schematic of zirconia AFR sensor



Figure 5 Zirconia AFR sensor

Here, I_p indicates the pumping current [A], n the number of electrical charge in electrode reaction (=4), F the Faraday constant, S the cross section area for gas diffusion pre, P the pressure, R gas constant, T the temperature, L the length of gas diffusion pore, D_{O_2} the oxygen diffusion coefficient, and C_{O_2} oxygen concentration.

It is possible to calculate the AFR as the excess/shortage of O_2 compared to the theoretical AFR is calculated by measuring the pumping current and thus the AFR is calculated.

Materials and Methods

Setup for fuel consumption measurement

Figure 6 shows the system setup for fuel consumption measurement. The ultrasonic flowmeter was connected behind the tail pipe of the test vehicle. The zirconia AFR sensor is capable of addressing simultaneous measurement at a close position and providing the two roles of exhaust gas flowmeter and fuel flowmeter in the same unit by installing it inside the piping for ultrasonic flowmeter. The exhaust side of the exhaust gas flowmeter was connected to the CVS system to implement correlation evaluation with carbon balance method.

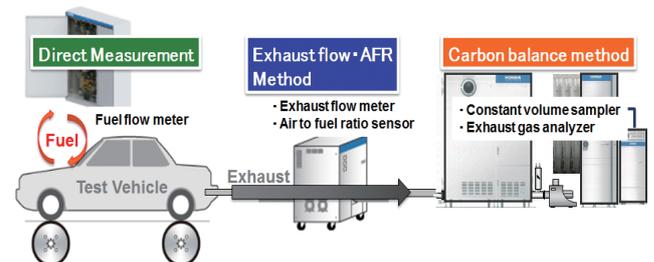


Figure 6 Experimental setup for fuel consumption measurement

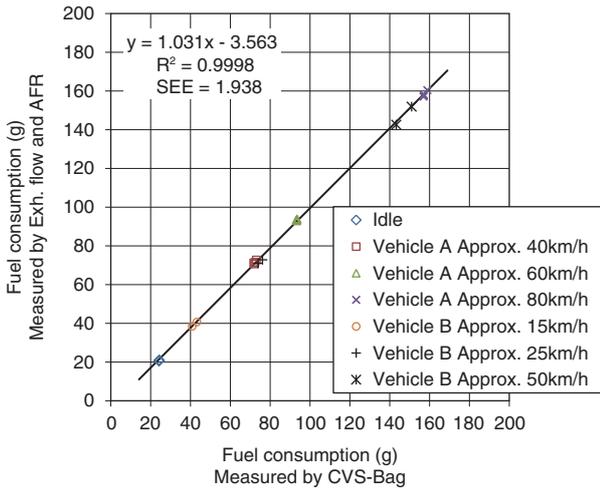


Figure 7 Correlation with CVS carbon balance method (integrated fuel consumption, steady-state condition)

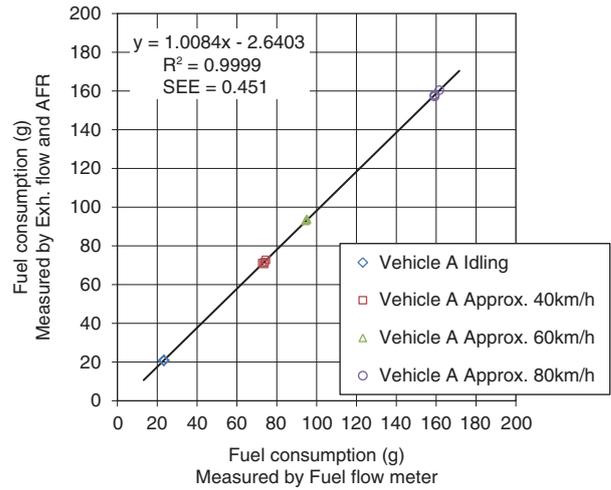


Figure 9 Correlation with fuel flow method (integrated fuel consumption, steady-state condition)

Results and Discussion

Comparison with carbon balance method (Integrated fuel consumption)

Figure 7 shows a comparison of fuel consumption calculated by exhaust gas flow rate/AFR method and carbon balance method (CVS bag measurement) in steady-state driving condition. The value of exhaust gas flow rate/AFR method in each plot shows the integrated value for continuous measurement for 3 minutes, and exhaust gas for the same block was collected in the bag with carbon balance method. This results show that there is a good correlation in the range from idling to driving at 80 km.

Figure 8 shows the integrated fuel consumption in each phase when FTP test cycle is run twice. The data



Figure 8 Comparison with CVS carbon balance method (integrated fuel consumption, transient test)

indicated by “n1” are for the first run and “n2” for the second run. In this figure, the difference in measurement results from those of carbon balance method indicated by “Difference” was approximately 2% at maximum. It also shows that the tendency for data between phases is reproduced well by n1 and n2.

Comparison with fuel flowmeter (Integrated fuel consumption)

Figure 9 shows a comparison with fuel flowmeter under steady-state driving condition. A similar method of data processing as Figure 7 was in each plot. As was in comparison with carbon balance method, there was good correlation over the entire range.

Figure 10 shows a comparison of cumulative fuel consumption in transition cycles used in Japan, Europe

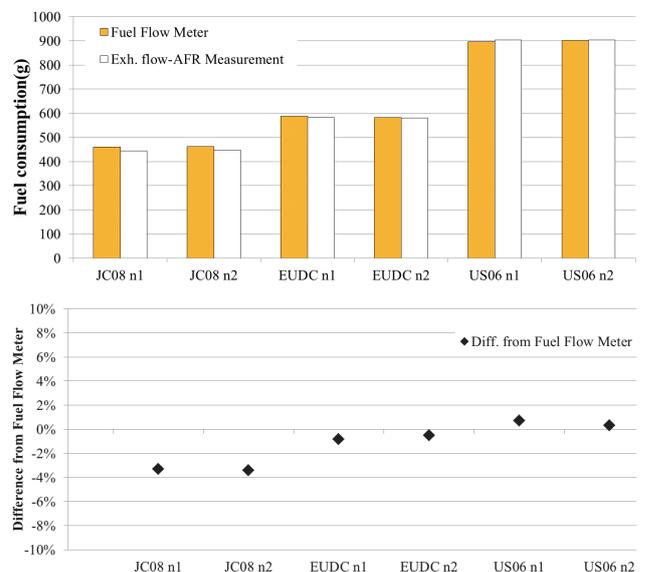


Figure 10 Comparison with fuel flow method (integrated fuel consumption, transient test)

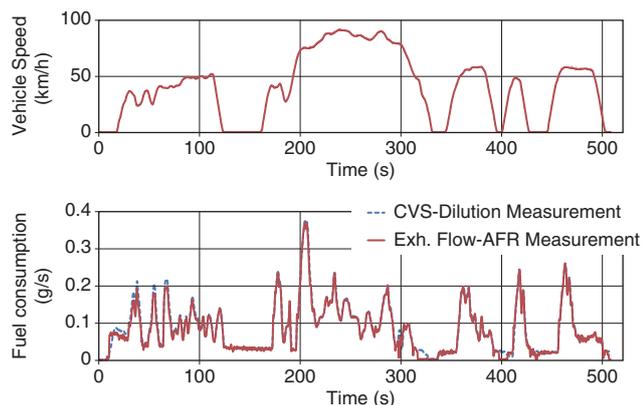


Figure 11 Example of real time fuel consumption measurement (FTP75 cycle phase 1)

and the U.S. The same tendency as shown in Figure 8 is observed.

Comparison with carbon balance method (Real-time fuel consumption)

Figure 11 shows the results of comparison on measurement of the real-time fuel consumption in the FPT75 cold start phase using the exhaust gas flow rate/AFR method and carbon balance method (dilute stream measurement). It is evident that the overall behavior matches relatively well. Figure 12 shows the enlarged figure for the time slot from 300 seconds to 500 seconds along with the AFR measurement results. There is a block for vehicle deceleration where AFR drops rapidly from around the theoretical AFR. This indicates that the fuel-cut mechanism is operating to reduce the unnecessary fuel consumption in deceleration. As expected, the measurement value for fuel consumption by exhaust gas

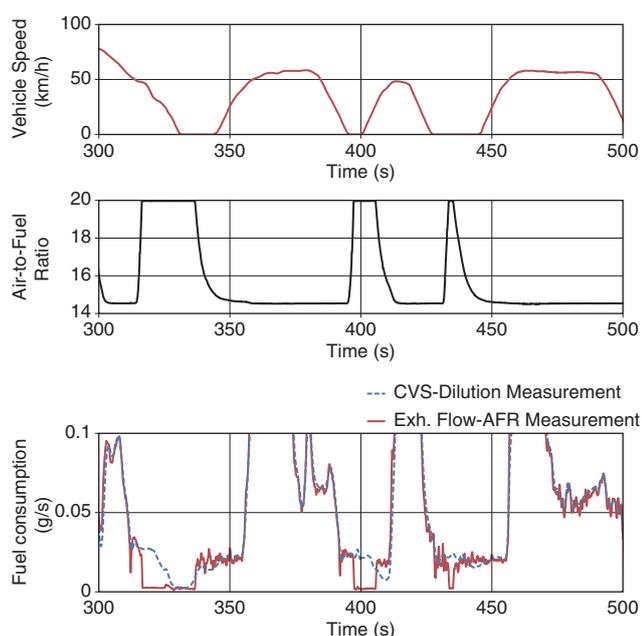


Figure 12 Real-time fuel consumption measurement of FTP75 phase 1

flow rate/AFR method is nearly zero during fuel-cut. On the other hand, there are blocks where fuel consumption does not reach zero even during fuel-cut when carbon balance method is used. It is considered that this is caused by the effect of delay in gas feeding in the sample line and the response time of exhaust gas analyzer. Based on this result, it is surmised that the exhaust gas flow rate/AFR method may be able to deliver the results with better accuracy when real-time measurement corresponding to the engine operation is required.

Conclusion

The greatest advantage of the exhaust gas/AFR method is that it can be used to measure safely and simply by only connecting the flowmeter to the exhaust pipe. It is expected to be used popularly in benchmark tests with unknown detailed characteristics for the vehicle and the actual fuel efficiency evaluation in completed vehicles. Furthermore, this method is one of the exhaust gas flowmeter applications. We would like to propose more applications in combination with other HORIBA products and contribute to the development of HORIBA and automotive industry.

References

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